

Theory of Structures

Notes by-

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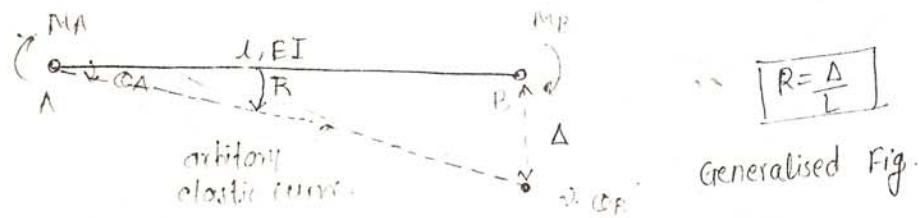
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[Slope Deflection Method] Displacement Mtd

slope deflection eqⁿ:

Sign conventions: All clockwise moments & rotations are positive. (+)

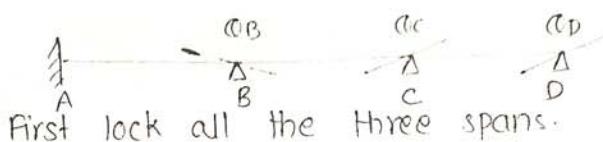


MA & MB are the end moments of an unloaded beam element.

When its ends rotate by θ_A & θ_B and the element itself rotates by Δ , the end moments & rotations are related through the eqⁿ:

$$\left. \begin{aligned} MA &= \frac{4EI\theta_A}{L} + \frac{2EI\theta_B}{L} - \frac{6EI\Delta}{L^2} \\ MB &= \frac{2EI\theta_A}{L} + \frac{4EI\theta_B}{L} - \frac{6EI\Delta}{L^2} \end{aligned} \right\}$$

The slope deflection eqⁿ are derived for a loaded beam element by adding these end rotations moments to the fixed end moments / locking mmt.



MA = Final mmt.



FEM Due to the actual loading

$$MA = (MA)_0 + \frac{2EI}{L} [2\theta_A + \theta_B - 3R]$$

$$MB = (MB)_0 + \frac{2EI}{L} [\theta_A + 2\theta_B - 3R]$$

$$R = \frac{\Delta}{L}$$

R is positive if clockwise.

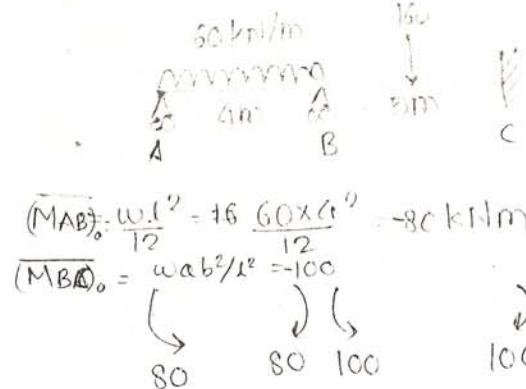
$$\boxed{\begin{aligned} MBA + MBC &= 0 \\ MCB + MCD &= 0 \\ MDC &= 0 \end{aligned}}$$

L on

epend

Prob. Analyse and plot BMD.

MPSC
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$$\frac{(MAB)_o}{EI} = \frac{w_1 l^2}{12} = 16 \cdot \frac{60 \times 80^2}{12} = -80 \text{ kNm}$$

$$\frac{(MBA)_o}{EI} = \frac{w_2 b^2}{12} = 100 \cdot \frac{100^2}{12} = +100 \text{ kNm}$$

SD eqⁿ for AB,

$$MAB = -80 + \frac{2EI}{4} (2\theta_A + \theta_B) = -80 + \frac{4EI}{4} \theta_A + \frac{2EI}{4} \theta_B$$

$$MBA = +80 + \frac{2EI}{4} (\theta_A + 2\theta_B) = 80 + 0.5EI\theta_A + EI\theta_B$$

$$MBC = -100 + \frac{2EI}{5} (2\theta_B + \theta_C) = -100 + 0.8EI\theta_B$$

$$MCB = +100 + \frac{2EI}{5} (\theta_B + 2\theta_C) = +100 + 0.4EI\theta_B$$

$$\therefore M_{BA} + M_{BC} = 0$$

$$\therefore -20 + \frac{2EI}{4} \theta_A + \frac{2EI}{5} (2\theta_B) + \frac{2EI}{5} (2\theta_B) = 0$$

$$\therefore -20 + 0.5EI\theta_A + EI\theta_B + 0.8EI\theta_B = 0$$

$$\therefore -20 + 0.5EI\theta_A + 1.8EI\theta_B = 0$$

$$\left. \begin{aligned} EI\theta_A + 0.5EI\theta_B &= 80 \\ 0.5EI\theta_A + 1.8EI\theta_B &= 20 \end{aligned} \right\}$$

$$\therefore \theta_A = \frac{86.45}{EI}$$

$$\theta_B = \frac{-12.9}{EI}$$

$$\therefore MAB = 0$$

$$MBA = 110.32$$

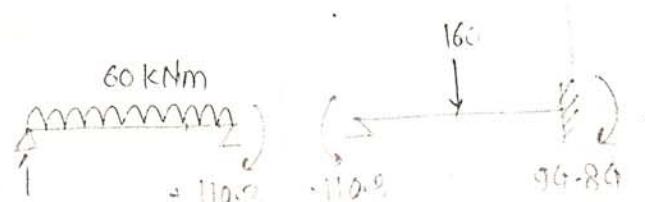
$$MBC = -110.32$$

$$MCB = +94.84$$



In case

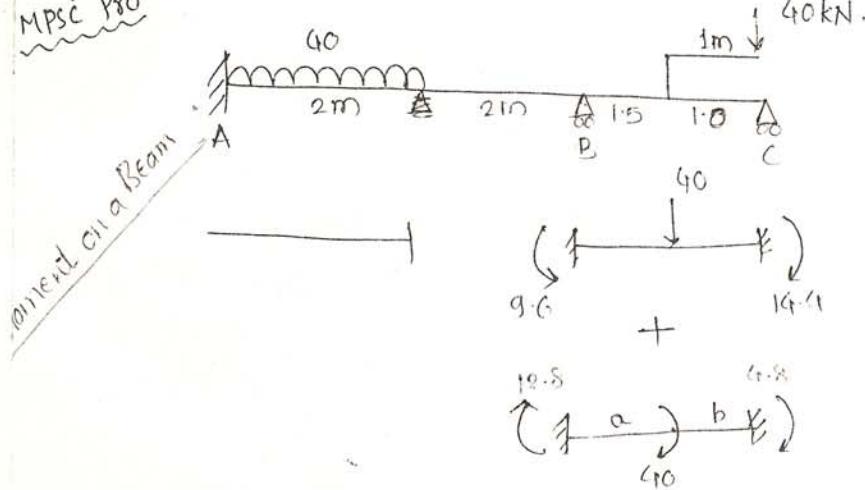
BMD
(Nature).



Magnitude

MPSCL PRO

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$$\overline{MBC}_1 = \frac{40 \times 1.5 \times 1^2}{2.5^2} = 9.6$$

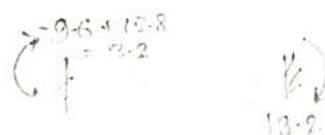
$$\overline{MCB}_2 = \frac{40 \times 1.5^2 \times 1}{2.5^2} = 14.4$$

$$\overline{MBC}_2 = \frac{40 \times 1}{2.5^2} [3 - 1] = 12.8$$

$$\overline{MCB}_2 = \frac{40 \times 1.5}{2.5^2} [2 - 1.5] = 4.8$$

Net

$$\therefore \overline{MBC} =$$



Q) xho case of overhang



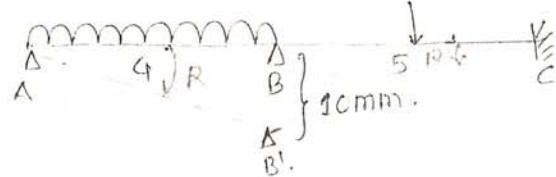
$$[MAB + MAD = 0]$$

$$mmmm)$$

$$\text{Reactive mmt. } MAD = \frac{MAD - \frac{wL^2}{2}}{2} \\ = \frac{60 \times 1.5^2}{2} = 67.5$$

In case of yielding of support :- In previous ex. yields by 10 mm.

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$$\therefore MAB = -80 + \frac{2EI}{4} \left[2\theta_A + \theta_B - \left(\frac{3 \times 0.01}{4} \right) \right]$$

$$MBA = +80 + \frac{2EI}{4} \left[2\theta_A + 2\theta_B - \left(\frac{3 \times 0.01}{4} \right) \right]$$

$$MBC = -100 + \frac{2EI}{5} \left[2\theta_B - 3 \times \left(\frac{-0.01}{5} \right) \right]$$

$$MCB = +100 + \frac{2EI}{5} \left[\theta_B - 3 \times \left(\frac{-0.01}{5} \right) \right]$$

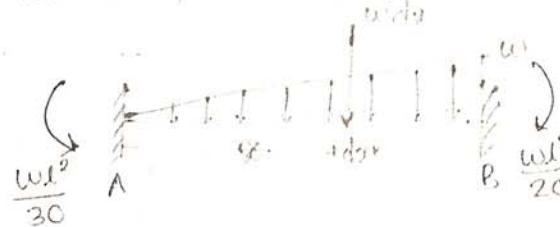
as R rotating clockwise

i.e. AB rotating clockwise

$\therefore +ve$

$$\text{eg:- } \frac{P a^2 b}{l^2} \left(\frac{l}{a+b} \right) \frac{P a^2 b}{l^2}$$

Find FEM for Triangular load -



$$\omega = \frac{w \cdot l}{l}$$

$$\text{Elementary mmt at A} = \frac{P \cdot a^2 b^2}{l^2}$$

$$\left(\frac{w x}{l} \right) dx \cdot x (l-x)^2$$

$$MA = \frac{\omega}{l^3} \int_0^l x^2 (l-x)^2 dx$$

$$MA = \frac{\omega}{l^3} \int_0^l x^2 (l^2 - 2lx + x^2) dx$$

$$= \frac{\omega}{l^3} \left[l^2 \cdot \frac{x^3}{3} - 2l \cdot \frac{x^4}{4} + \frac{x^5}{5} \right]_0^l = \frac{\omega}{l^3} \left[\frac{l^5}{3} - \frac{l^5}{2} + \frac{l^5}{5} \right]$$

$$\boxed{MA = \frac{\omega l^2}{20}}$$

$$MB = \frac{\omega}{l^3} \int_0^l x^3 (l-x) dx$$

$$\boxed{MB = \frac{\omega l^2}{20}}$$

is
much
SDM

When
some
beams

consider
SDM

Magnitude -